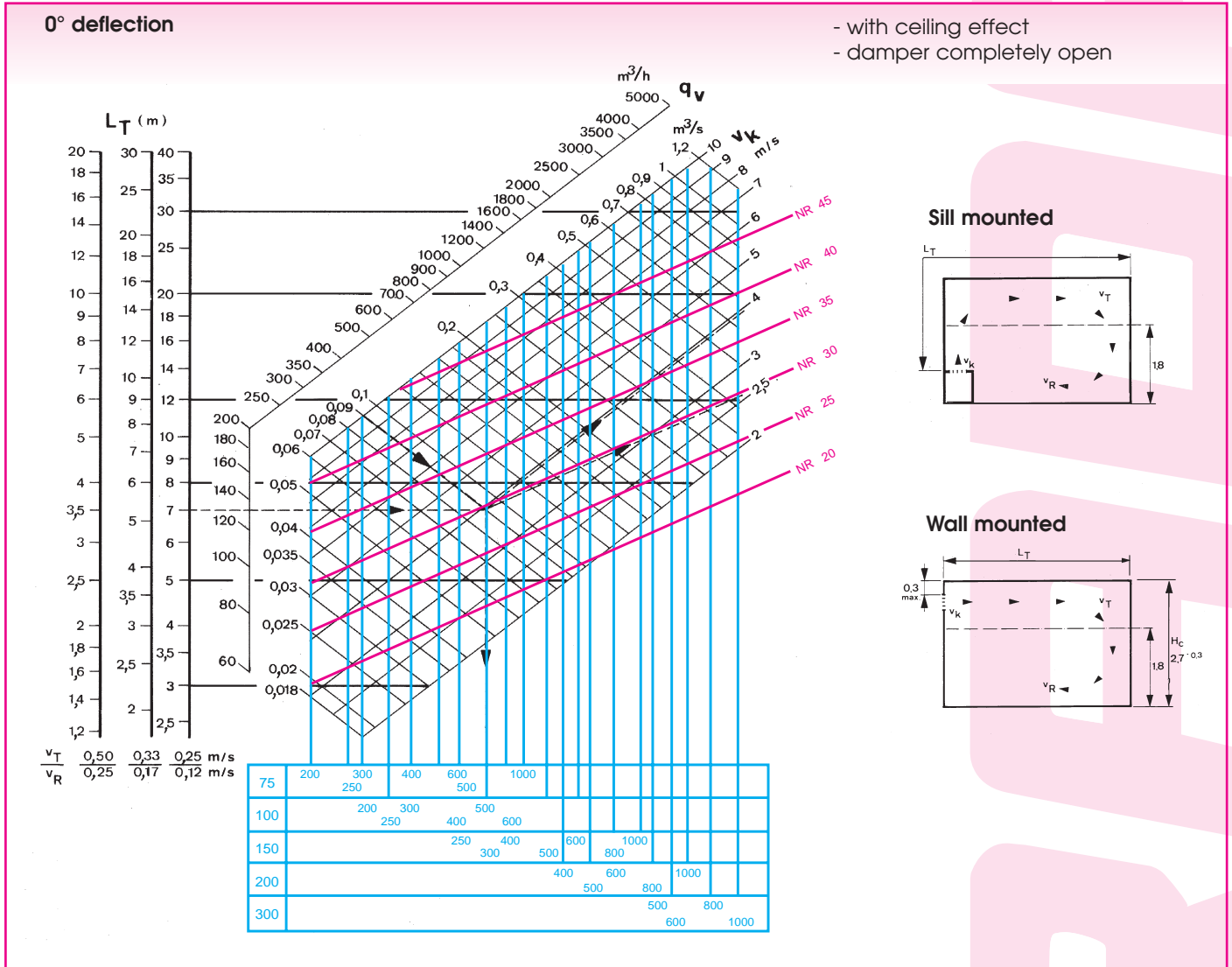
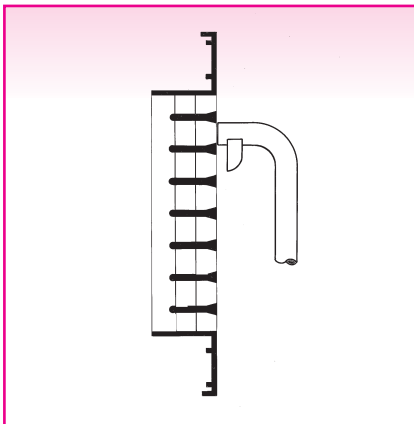


Selection diagram - supply



**Air flow rate measurement-
supply**



Velometer jet 2220 A or 6070

A_k-values (m²)								
H (mm)	L (mm)							
	200	250	300	400	500	600	800	1000
75	0,006	0,008	0,009	0,013	0,016	0,019	0,027	0,031
100	0,009	0,011	0,013	0,019	0,023	0,027	0,038	0,047
150	—	0,019	0,023	0,031	0,038	0,047	0,063	0,078
200	—	—	—	0,042	0,053	0,063	0,084	0,108
300	—	—	—	—	0,084	0,099	0,133	0,167

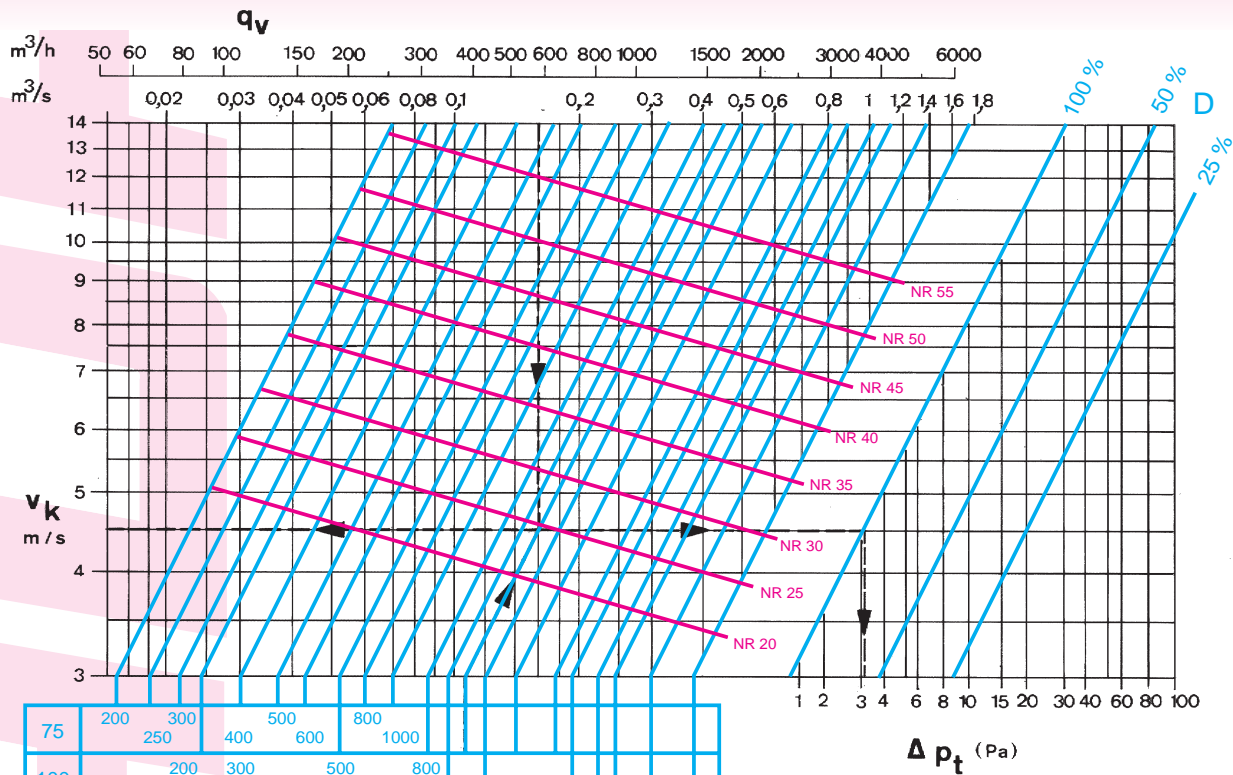
Correction factors:

- Throw correction factor without ceiling effect

Distance between ceiling and supply grille	Correction
≥ 0,9 m	L _T X 0,75

- Correction factors for vertical vane deflection of flow equalizer (see p. 1 231 verso)

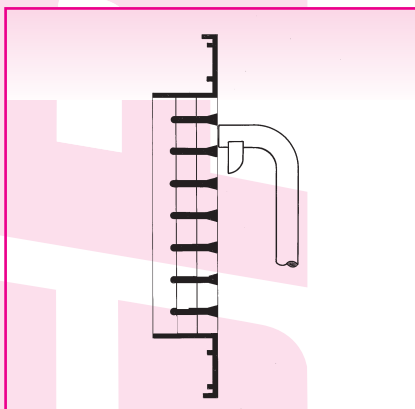
Selection diagram - exhaust



75	200	300	400	500	600	800	1000		
100		200	300	400	500	600	800		
150			250	400	500	600	800		
200				300	400	500	600	800	
300					400	500	600	800	1000

When 15° deflected bars are used, air flow rate will be reduced by 5% at listed Δp_t and NR values.

Air flow rate measurement - exhaust

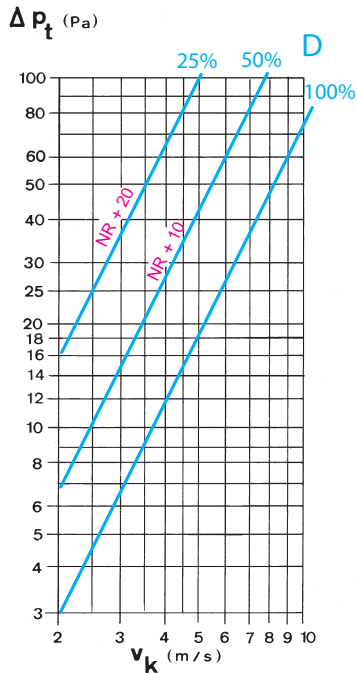


Velometer jet 2220 A or 6070

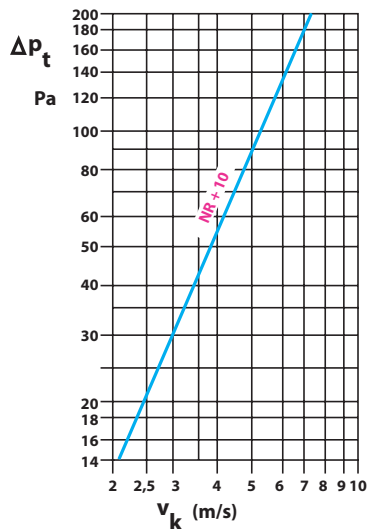
H (mm)	A_k -values (m^2)							
	L (mm)							
	200	250	300	400	500	600	800	1000
75	0,005	0,006	0,007	0,010	0,012	0,014	0,020	0,023
100	0,007	0,008	0,008	0,014	0,017	0,020	0,028	0,035
150	—	0,014	0,017	0,023	0,028	0,035	0,047	0,058
200	—	—	—	0,031	0,039	0,047	0,063	0,080
300	—	—	—	—	0,063	0,074	0,099	0,125

Pressure loss - supply

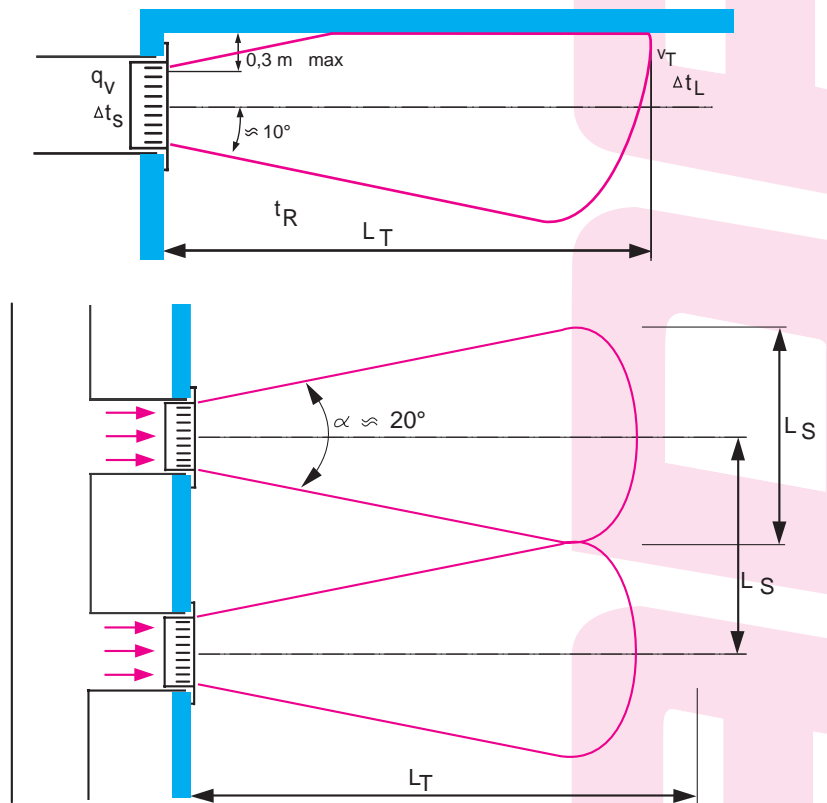
with damper type .. 7



with perforated sheet type .. 3



Example



SUPPLY

Selection data:

Air flow rate $q_v = 0,09 \text{ m}^3/\text{s}$
 Throw $L_T = 7 \text{ m}$ at $v_T = 0,25 \text{ m/s}$.

Solution:

Grille 500×100 or $300 \times 150 \text{ mm}$.
 Supply air velocity $v_k = 3,9 \text{ m/s}$.
 Noise level NR 29
 Total pressure loss with perforated sheet: $\Delta p_t = 59 \text{ Pa}$.
 Noise level correction NR
 $29 + 10 = \text{NR } 39$

EXHAUST

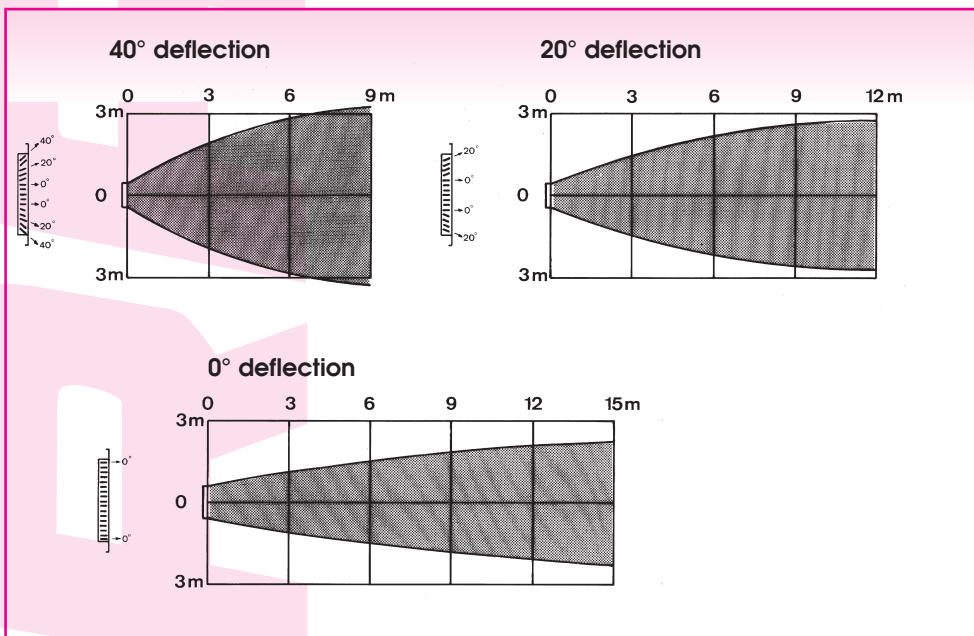
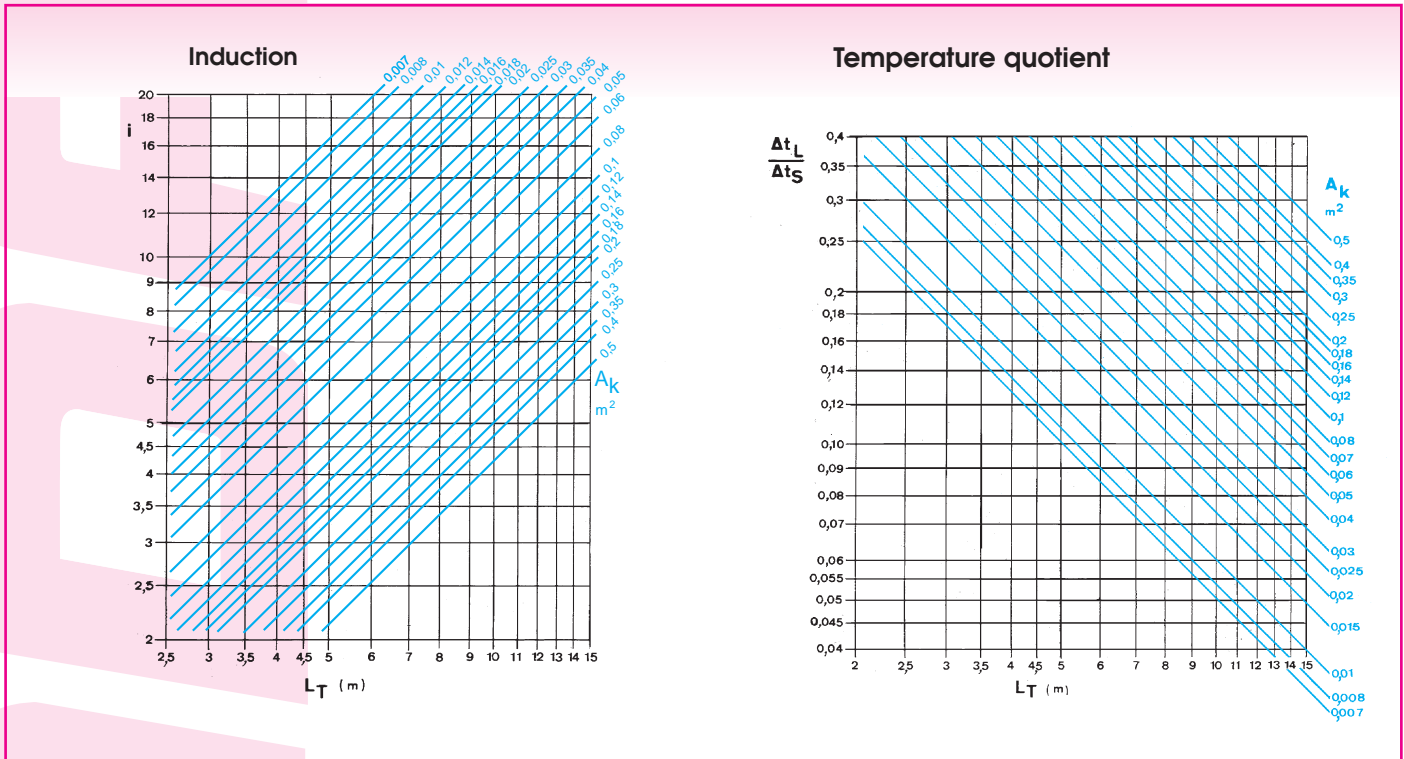
Selection data:

Exhaust air flow rate $q_v = 0,16 \text{ m}^3/\text{s}$

Solution:

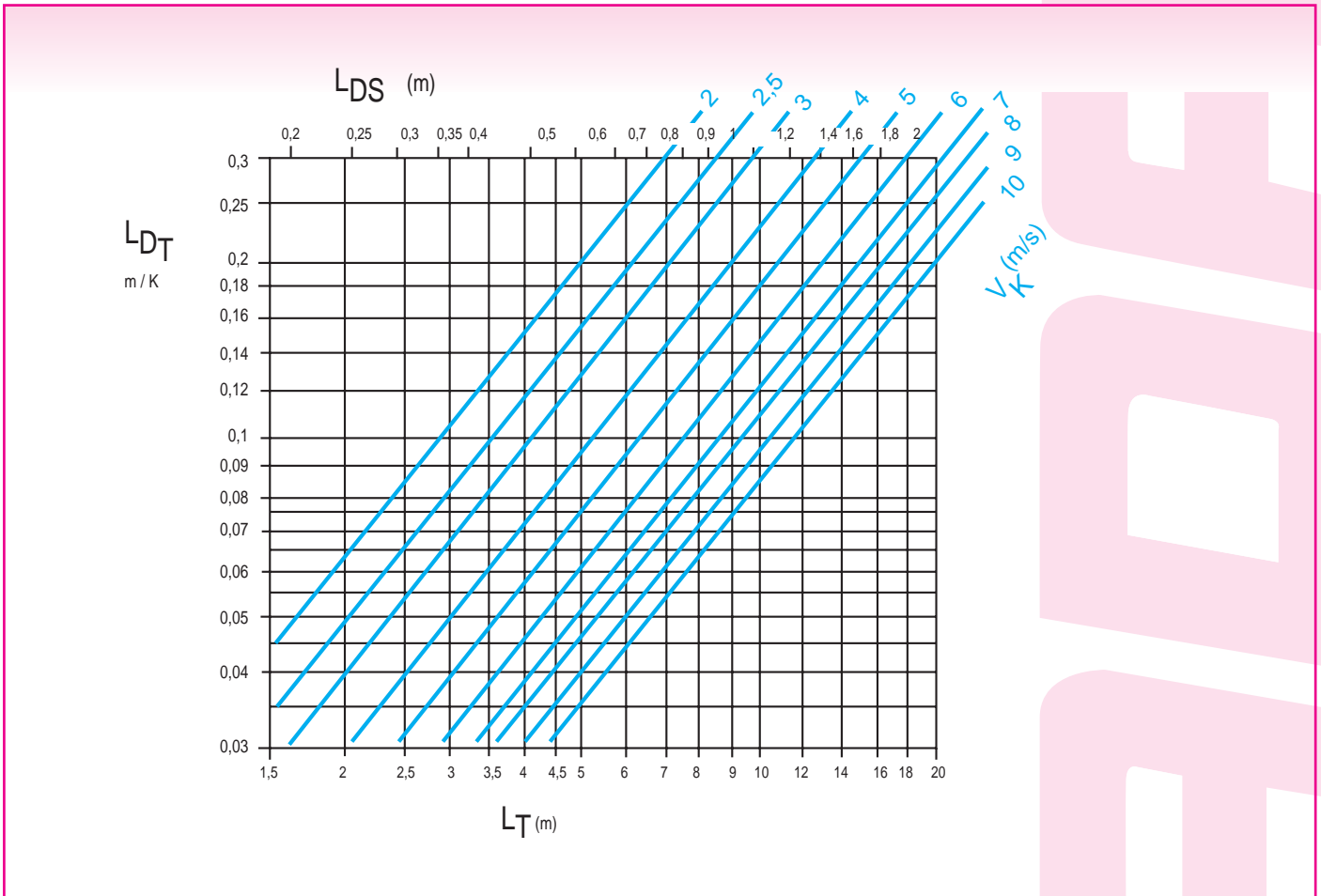
Grille $1000 \times 100 \text{ mm}$.
 Air velocity $v_k = 3,9 \text{ m/s}$.
 Noise level NR 25
 Total pressure loss with damper
 100 % open: $\Delta p_t = 3,2 \text{ Pa}$

Induction and temperature quotient with ceiling effect (also valid for linear grilles)



Correction factors Correction factors for vertical vane deflection of flow equalizer	Type	Deflection	A_k	v_k	L_T	NR	i	$\frac{\Delta t_L}{\Delta t_S}$
	300	20°	x 0,87	x 1,15	x 0,85	+ 3	x 1,4	x 0,7
	40°	x 0,80	x 1,25	x 0,75	+ 5	x 2	x 0,5	

Drop requirements



Drop requirements

The total drop is the maximum vertical distance between the centre of a grille core and the lower point of a specified envelope, determined by the envelope velocity v_T .

The total drop consists of two elements: $L_D = L_{DS} + L_{DT}$

- 1) The isothermal drop L_{DS} is the distance between the centre of an air current and the lowest point of the envelope.
- 2) The non-isothermal drop L_{DT} is the distance between the centre of the grille core and the air current centre line, at the place of measurement.

